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VERIFICATION OF TRANSLATION

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Declare as follows:

1. That I am well acquainted with both the English and German languages, and
2. That the attached document is a true and correct translation made by me to the best of my knowledge and belief of:
 - a) Patent Specification WO 2005/021915

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DRIVE SYSTEM FOR REGULATING DEVICES IN MOTOR VEHICLES

15 **Description**

The invention relates to a drive system for regulating devices in motor vehicles according to the preamble of claim 1.

20 From WO 01/89063 A1 a drive unit is known for regulating devices in motor vehicles having the features of the preamble of claim 1. Since with this known drive unit at least one component part of the disc armature motor such as for example the drive shaft, the magnetic short circuit or the housing shell on the motor side also undertakes an additional mechanical function of the planet wheel transmission and/or
25 a mechanical component part such as the housing shell on the gearing side undertakes a function of the disc armature motor the drive unit is as a result of the multiple use of individual component parts characterised by a minimum number of parts and minimum weight as well as following the interfitting box structure and thus the high degree of integration of the component parts of the drive unit by a small
30 volume and more particularly by its flat method of construction.

As opposed to the drive units normally used comprising rod armature motors with worm gearing as reduction gearing which through their type of construction are not equally suited to right and left hand vehicle doors where the structural height in the
35 door region is an interference and whose installation position is not flexible particularly through rotation of the drive unit, wherein the centre of gravity of the

motor lies outside of the screw connection of the drive unit and whose system mass is very high, the drive unit according to WO 01/89063 A1 is particularly suitable for installation in vehicle doors as a result of its properties previously mentioned since it can be incorporated without problem in a base part, door module or a support plate
5 in any angular position and independently of which door side and provides a single assembly plane and thus a variable use.

A further reduction in the structural volume of the drive unit known from WO 01/89063A1 comes up against stability limits if the material thickness of the
10 component parts of the drive unit used is reduced and the distances between the component parts which are already boxed in each other with a high degree of integration are reduced. Thus the small air gap between the armature disc and the permanent magnet of the stator of the disc armature motor in the case of a stable mounted armature disc which is not permanently precisely aligned would lead to
15 contact between the armature disc and permanent magnets and thus to premature wear and breakdown of the drive unit. Furthermore the mechanical stability of the planet wheel transmission would be endangered if the centring of the mechanical component parts is not permanently guaranteed and also under severe strain no deformations of the gearing parts occur.

20 The object of the present invention is therefore to provide a drive system for regulating devices in motor vehicles of the type mentioned at the beginning which ensures a further reduction in the structural volume and more particularly an extremely flat method of construction without impairing the functioning, mechanical
25 stability and durability of the drive unit.

This is achieved according to the invention through the features of claim 1.

The solution according to the invention enables the production of a drive system for
30 regulating devices in motor vehicles with minimum structural volume and more particularly an extremely flat drive system without having to take into consideration deterioration in the functioning, mechanical stability and durability of the drive system.

35 Thus the drive system according to the invention with its low structural height, reduced weight and reduced costs achieved through material savings and multi-

functioning of the component parts used is particularly suitable for electro-motorised regulating devices in motor vehicles where the installation volume available is small and where when installed in motor vehicle doors or in roof structures a minimum structural height and variable usability is required.

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The solution according to the invention is based on the knowledge that an exact bearing as well as axial guidance and alignment of the component parts of the drive system is of elementary importance for the mechanical stability of the drive system and a reduction in the distances between the component parts as well as a reduction 10 in the weight and thickness thereof. By fitting the fixed drive axle, journal of the output internal geared wheel and drive hub cylinder of the drive hub onto each other as a result of the length of the bearing which can be achieved despite the compressed construction an accurate and permanent guidance and alignment of the armature disc as well as of the journal of the slow running output internal geared 15 wheel is achieved which is thus securely mounted and guided on the entire length of the fixed drive axle in order to avoid tumbling movements.

The length of the drive hub cylinder preferably corresponds substantially to the height of the rollers so that for guiding and aligning the armature disc the entire length or 20 structural depth of this part of the planet wheel transmission is used which is required in any case for attaching and guiding the radially flexible ring which meshes both with the internal gearing of the hollow wheel fixed on the housing and also with the internal gearing of the output hollow wheel.

- 25 In order to secure the position and alignment of the fixed drive axle this is connected to a first housing cover of the housing surrounding the drive element of the regulating device, the armature disc motor and the planet wheel transmission, and is supported on a second housing cover of the housing.
- 30 By selecting matching materials in the region of the bearing point between the fixed drive axle, journal of the output internal geared wheel and drive hub cylinder of the drive hub whereby the fixed drive axle and the drive hub cylinder are made of steel or a steel alloy and the journal is made of sintered metal, optimum bearing properties are achieved. It is thereby further possible to use the high-grade and comparatively

expensive sintered metal bearing material only once and thereby to produce both an optimum bearing for the fixed drive axle and for the fast-turning drive hub.

A further important measure for reducing the volume and more particularly structural

5 depth of the drive system lies in connecting the hollow wheel fixed on the housing to a base disc or to make it part of a base disc which supports the permanent magnets of the disc armature motor and has centring means provided around the periphery to centre at least one of the two housing covers relative to the base disc.

10 The base disc centres as a central positioning element at least the one but preferably both housing covers of the housing and at the same time supports the hollow wheel gearing which is either formed integral on the base disc or is connected to the base disc in a two-part variation as plastics or metal parts in the twin-component system so that the base disc without any increase in the structural depth of the drive system
15 fulfils a double function as gearing element and as centring element for the precision alignment and bearing of the component parts of the drive system.

In order to increase the mechanical stability of the base disc this can be provided

with a cropped peripheral edge as well as for positioning and securing the position of

20 the permanent magnets with profiling.

Furthermore the base disc can be connected to a disc, preferably of plastics, supporting the permanent magnets through stamped areas and/or cropped bracket plates which are stamped or bent out from the base disc.

25

Socket areas can also be incorporated into the outwardly directed edge of the hollow wheel fixed on the housing and are adapted at least in sections to the contour of the permanent magnets of the disc armature motor so that the permanent magnets can be readily positioned there.

30

More particularly the socket areas can be formed in the outwardly directed edge of the hollow wheel fixed on the housing as radially outwardly opening sockets.

The base disc preferably consists of plastics in which socket areas are incorporated which are adapted at least in sections to the contour of the permanent magnets of the disc armature motor.

- 5 The socket areas of the hollow wheel fixed on the housing of the base disc surround the permanent magnets so far that a defined position of the permanent magnets is secured.

Determining the position of the permanent magnets is formed at the same time
10 through the hollow wheel which is fixed on the housing and made from plastics or a paramagnetic material for simplifying assembly. Fixing the permanent magnets is carried out by shaping the internal wheeled gearing or through fixing elements for subsequent positive locking or force locking connection of the base disc to the permanent magnets which are incorporated in the plastics.

15 The permanent magnets themselves are more particularly made by injection moulding process from plastics-bonded high-energy materials with their possibilities for shaping plastics parts. Flatness and tolerances are ensured through the tool to a high extent so that the base disc can be used in connection with the plastics-bonded
20 magnet system without parasitic magnetic losses as short-circuit. Sticking and fixing the permanent magnets is thereby unnecessary because the magnet material and base disc can be connected to each other with undercut sections.

In order to be able to save mass the ferrite metal parts can be thickened by tailored
25 blanks only in the region of the flux-conveying short circuits.

Likewise to increase the stability of a very thin drive hub this can have several cropped angles formed out of its surface and aligned towards the side of the rollers and thus use space which is available in any case thereby not causing any increase
30 in the structural depth. The requirement is thereby met that the pair of rollers preferably mounted on sliding or rolling bearings can be placed asymmetrically, leading to a further reduction in the structural height.

In order to simplify the manufacture and material saving the drive hub cylinder and/or
35 the roller bearings can be formed as passages in the drive hub and the outer cylinder

face of the passages can be supported by hardened steel bushes with collar.

For a better guidance of the radially flexible ring the rollers are preferably provided with grooves.

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In order to absorb force peaks when the regulating device moves into an end position such as for example the upper and lower stops of a window lifter, circumferentially active damping elements are integrated in the output internal geared wheel of the drive system. The damping is thereby shifted in the drive system radially further out
10 in to the region of the connection and the drive system can be fixed by way of example in a door module by means of a bayonet connection.

As an alternative or in addition the output internal geared wheel can be connected to a loop spring brake whereby the self locking action of the drive system is increased
15 and by way of example in the case of a window lifter unauthorised access to inside the vehicle by forcing down the window pane is thereby prevented.

A steel ring can optionally be provided in the output internal geared wheel to support the radial forces which act on the output internal geared wheel. This steel ring can
20 additionally undertake the function of the brake ring for the installation of the loop spring.

Furthermore a sintered metal / plastics connection can be provided in the output internal geared wheel.

25

The drive element of the regulating device, for example a cable drum of a window lifter, can either be connected in the axial direction to the output internal geared wheel and be axially fixed through holding clips integrated in the second housing cover or the output internal geared wheel is integrated or moulded into the drive
30 element of the regulating device.

The first alternative provides a service solution where in the event of breakdown or wear the cable drum can be easily exchanged without replacing the motor/gearing unit whilst with the second alternative the enhanced integration produces an ultra-flat
35 structural form of the drive system with integrated drive element of the regulating device.

The invention and the idea on which the invention is based will now be explained in further detail with reference to the embodiments illustrated in Figures 1 to 3 of the drawings. They show:

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Figure 1 a perspective exploded view of the structural and functional parts of an ultra-flat drive system according to the invention for a cable window lifter;

10 Figure 2 a longitudinal sectional view through the drive system according to Figure 1 and

Figure 3 a longitudinal sectional view through a drive system for a cable window lifter with a cable drum integrated in the output internal geared wheel.

15

Figure 1 shows an exploded view of a drive system illustrated in longitudinal sectional view in Figure 2 for a cable window lifter of a motor vehicle. The drive system comprises a disc armature motor 3, a planet wheel transmission 5, 6, 21, 71, 72 and a cable drum 10 which are mounted in a housing which comprises two 20 housing covers 41, 42 and is centred on a base disc 2.

The electronically commutated disc armature motor 3 in this embodiment comprises an armature disc 31 and several permanent magnets 32 arranged spaced out round the circumference of the base disc 2 (Figure 2). The armature disc 31 is connected 25 to a drive hub 6 which is preferably formed as a stamped steel part and has a passage formed as a drive hub cylinder 60 as well as two passages 61, 62 for two roller bearings 63, 64 on which two rollers 71, 72 of the planet wheel transmission are mounted and supported through rolling bearings 65 opposite the rollers 71, 72. Furthermore several cropped angles 66 are provided on the drive hub 6 for 30 increasing the stability of the drive hub 6.

As an alternative to the drive hub 6 designed as a stamped steel part the drive hub 6 can be made from sintered metal or high-grade glass-fibre or carbon-fibre reinforced plastics or combinations of these materials. The outer cylindrical face of the 35 passages for the drive hub cylinder 60 and for the roller bearings 61, 62 are more particularly supported through hardened steel bushes with collar. The drive hub

cylinder 60 has a length which corresponds roughly to the height of the rollers 71, 72 so that a maximum guide length is guaranteed for the drive hub cylinder 60.

The planet wheel gearing is comprised of a hollow wheel 20 fixed on the housing and
5 having internal toothing 21 with a first number of teeth, an output hollow wheel 5 with
internal toothing 51 with a second number of teeth and a radially flexible ring 8 on the
inner sleeve face 81 of which the rollers 71, 72 roll and whose external toothing 82
meshes both the internal toothing 21 of the hollow wheel 20 fixed on the housing and
with the internal toothing 51 of the output wheel 5. The radially flexible 8 is guided
10 for axial security in peripheral grooves 73 of the rollers 71, 72.

The hollow wheel 20 which is fixed on the housing is connected to the base disc 2 or
is formed as a part of the base disc 2. The base disc 2 serves in addition to securing
the position of the permanent magnets 32 of the disc armature motor 3 also as a
15 central positioning element for centring the two housing covers 41, 42 and has for
this purpose centring elements 91, 92 which are designed as studs and indentations
and which correspond with the corresponding counter centring elements 93, 94 of
the first housing cover 41. The base disc 2 forms together with the first housing
cover 41 at the same time the magnetic short circuit for the disc armature motor 3.

20 The first housing cover 41 has to increase its stability additional cropped angles as
well as the swages and indentations whilst the second housing cover 42 is aligned
and fixed on a flange formed on the base disc 2 or on a corresponding centring
recess or centring groove.

25 The base disc 2 is formed as a flat ring with a peripheral cropped angle 22 and can
be connected through indentations and/or cropped bracket plates to a disc preferably
of plastics which has sockets for the permanent magnets 31 of the disc armature
motor 3.

30 The internal toothing 21 of the hollow wheel 20 fixed on the housing can either be
formed in a metal edge of the base disc 2 or can be connected in a two-part variation
to a hollow wheel toothing of plastics or a metal component in the twin component
system. In this embodiment with for example a plastics hollow wheel this part of the
35 base disc 2 has open or closed socket bays for the permanent magnets 32 and the
circumferential forces acting on the plastics hollow wheel are transferred through

force locking connection with the cropped edge 22 of the base disc 2 where applicable additionally through the bracket plates which are bent and stamped out on alternate sides.

- 5 The output hollow wheel 5 is formed in two parts in the embodiment illustrated in Figure 2 and is comprised of an output drum 52 preferably of steel and a plastics ring 53 supporting the internal toothing 51. On the periphery of the plastics ring 53 there are two dampers 56 on each side of the peripheral webs 55 to correspond with stops 57 arranged in the circumferential direction on the cylindrical edge of the output drum
- 10 52 and forming damping members to absorb force peaks which result from the window lifter moving into the end positions.

The output hollow wheel 5 is connected to a journal 50 which extends over an essential part of the length of a fixed drive axle 40 which connects the two housing covers 41, 42 together, is pushed onto the fixed drive axle 40 and supports the drive hub cylinder 60 so that the journal 50 is mounted between the fixed drive axle 40 and the drive hub cylinder 60. The material pairing in the region of the bearing points between the fixed drive axle 40, the bearing journal 50 and the drive hub cylinder 60 is thus selected so that optimum bearing properties are achieved. More particularly 20 the journal is made from a sintered metal whilst the fixed drive axle 40 and the drive hub cylinder 60 are made from steel. It is thereby possible to insert the higher-grade and more cost-intensive sintered metal bearing material between the two layers of steel and to have to use this only once so that an optimum bearing is produced both for the fixed drive axle 40 and for the fast-turning drive hub cylinder 60.

- 25 The bearing of the journal 50 over practically the entire length of the fixed drive axle 40 ensures the centring of the output hollow wheel 2 so that no tumbler movements and subsequent noises occur and at the same time it forms a long axial bearing for the drive hub cylinder 60 so that the armature disc 31 which is mounted with slight 30 axial play relative to the base disc 2 and first housing cover 41 is likewise securely supported whilst maintaining a minimum air gap in respect of the housing cover 41 and the base disc 2.

The journal 50 has a pinion gearing 54 which engages through an opening in the output drum 52 and serves to receive internal toothing 12 of the cable drum 10. The cable drum 10 is connected through holding clips 22 to the second housing cover 42.

- 5 This produces a cost-effective service solution since in the event of wear the cable drum 10 can be removed from the drive system after removing the second housing cover 42 and then replaced.

Whereas the drive system illustrated in Figures 1 and 2 has nevertheless an overall thickness of for example about 35 mm despite a high degree of integration and the inter-boxed structural elements taking into account a high strength and durability as a result of the cable drum 10 fitted on the output internal geared wheel 5, the drive system illustrated in longitudinal section in Figure 3 is designed ultra-flat by optimizing all the possibilities for multiplexing and multi-use of component parts of the drive system whilst ensuring exact guidance of the rotating component parts.

The construction and function of the ultra flat drive system illustrated in Figure 3 corresponds as regards the housing, the disc armature motor and planet wheel transmission to the drive system according to Figure 2 with the measure that in this embodiment the cable drum 15 is integrated into the output internal geared wheel 5, i.e. the cable drum 15 forms at the same time the output internal geared wheel 5 and supports on its hollow cylindrical edge the internal toothing 51 which meshes with the toothing 82 of the radially flexible ring 8. Through this additional measure the structural depth of the ultra-flat drive system illustrated in Figure 3 can be reduced to about 20 to 23 mm and is particularly suitable therefore for fitting into vehicle doors or roofs where the system means that only a small structural depth is available.

The drive system illustrated in Figures 1 to 3 makes use of the principle of a "harmonic-drive-gearing". According to this principle the toothing 82 of the radially flexible ring 8 meshes both with the internal toothing 21 of the hollow wheel 20 fixed on the housing and with the internal toothing 51 of the output hollow wheel 5 which have a different number of teeth. The rollers 71, 72 which roll on the cylinder sleeve face 81 of the radially flexible ring 8 engage at two opposite points on the radially flexible ring 8 and deform this elliptically. The toothing 82 of the radially flexible ring 8 is thereby pressed into the cylindrical internal toothings 21, 51 and as a result of the different number of teeth of the internal toothings 21, 51 causes a permanent

continuous off-setting of the interengaging circumferential sections so that one complete revolution of the drive hub 6 only engenders a further movement of the output hollow wheel 5 by the difference in the number of teeth of the internal toothings 21, 51 of the hollow wheel 20 fixed on the housing and the output hollow wheel 5. A very high reduction can thereby achieved with this planet wheel gearing according to the principle of a harmonic-drive gearing.

LIST OF REFERENCE NUMERALS

- | | |
|----|----------------------------------|
| 2 | Base disc |
| 3 | Disc armature motor |
| 5 | 5 Output internal geared wheel |
| 6 | Drive hub |
| 8 | Radially flexible ring |
| 10 | Cable drum |
| 11 | Holding clips |
| 10 | 12 Internal toothed |
| | 15 Cable drum |
| | 20 Hollow wheel fixed on housing |
| | 21 Internal toothed |
| | 22 Peripheral cropped angle |
| 15 | 31 Armature disc |
| | 32 Permanent magnets |
| | 40 Fixed drive axle |
| | 41, 42 Housing cover |
| | 43 Cropped angle |
| 20 | 50 Journal |
| | 51 Internal toothed |
| | 52 Output drum |
| | 53 Plastics ring |
| | 54 Pinion gearing |
| 25 | 55 Peripheral webs |
| | 56 Dampers |
| | 60 Drive hub cylinder |
| | 61, 62 Passages |
| | 66 Cropped angles |
| 30 | 71, 72 Rollers |
| | 81 Inner sleeve face |
| | 82 External toothed |
| | 91, 92 Centring elements |
| | 93, 94 Counter centring elements |